# **Force acting on the Vertical Structure during Overflow Session P25 Session P25**



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- Overflow scenarios introduce complex challenges related to jet trajectories and air entrainment, demanding meticulous study.
- Existing analyses frequently overlook essential factors, especially sub-atmospheric pressure in the cavity, which can jeopardize structural stability.
- Moreover, current research reveals inaccuracies in calculating downstream water forces due to assumption of hydrostatic pressure, neglecting the influence of significant air bubble movement in air-water flows

- Investigate the intricate fluid dynamics of air-water flow and cavity pressure properties during vertical structure overflow.
- Utilize experiments to quantify the force acting on the model vertical structure, using different water pressure profiles: 1) Hydrostatic, 2) Bernoulli, 3) Discretized Navier-Stokes.
- Utilize the Buckingham Pi theorem to establish non-dimensional relationships among variables for convenient application of the research findings.

#### **▶ Research necessity**

### **▶ Research objectives**

The pattern of air cavity pressure and air-water pressure may differ with changes in the downstream flow regime conditions.



# **1. Introduction**

2. In subcritical and supercritical downstream scenarios, there exist distinct relationships between air cavity pressure and independent variables.

200

300

Pressure (Pa)

tmospheric pressure

- The number of total experimental cases: 192 cases with different independent variables.
- Independent variables and dependent variables are detailed in Table 1.

3. In overflow scenarios like this study, it has been verified through comparisons with measured data that to accurately determine the pressure acting on the downstream wall, the Navier-Stokes equation must be utilized.

- **▶ Background review: air entrainment mechanism (with Aerated and Non-aerated cases)**
- As shown in Figure 1, according to Chanson (1996), air bubble entrainment occurs along the cavity interface and at the end of the cavity through a re-entrant jet mechanism, leading to negative pressure in the air cavity.

# **2. Key Concepts**

Applying hydrostatic conditions is incorrect due to air bubble movement. In this study, we calculate and integrate 1) Hydrostatic, 2) Bernoulli, and 3) Navierstokes pressure, respectively, to determine the forces acting on the model

[Figure 1] **(Left)** Air entrainment mechanism **(Center)** Aerated case, **(Right)** Non-aerated case Reference: Chanson, H. (1996). *Air bubble entrainment in free-surface turbulent shear flows*. Elsevier.



 $h_{d,sub}$ 

**Supercritical Downstream** 

# **▶ Pressure acting on the structure**

- Upstream: Hydrostatic Pressure
	- Downstream: Cavity pressure + Navier Stokes



#### **▶ Downstream flow regime**

**Subcritical Downstream** 

[Figure 2] Pressure Profile distribution

# **3. Research Methodology**

[Figure 3] Downstream flow regime (**Left**: Subcritical, **Right**: Supercritical)

#### **▶ Experimental setup: Flume and model weir**

- As shown in Figure 1, air bubble entrainment occurs along the cavity interface and at the end of the cavity through a re-entrant jet mechanism, leading to negative pressure in the air cavity.

- All data, including cavity pressure, air-water pressure, discharge, water depth, velocity, and more, were collected over a 30 second duration. To facilitate visualization, the sampling rate was standardized to 10 Hz.
- There are noticeable differences between an aerated case and a non-aerated case, particularly in terms of cavity pressure.



# **4. Research Results**

### **5. Conclusions**

There are significant differences between the aerated and non-aerated cases in terms of factors such as air cavity pressure, tailwater depth, and the forces acting on the box.

400

500



[Figure 4] **(Left)** Side view of the flume that used in the experiment, **(Right)** Downstream face of the model weir

Bernoulli and Navier-Stokes pressures tend to align, but there are differences compared to hydrostatic pressure. This suggests that the velocity term significantly influences pressure in this sheet flow.

 $0.25$ 

 $0.20$ 

는 0.15

 $0.10$ 

 $0.05$ 

 $0.00$ 

You can access a real-time pressure data video demonstration through a QR code (Top: Aerated, Bottom: Non-aerated).



### **▶ Experimental variables**



#### **▶ Mathematical expression for different pressure profiles**

- Find the pressure profile that best matches the measured force.
- For Hydrostatic condition: Assuming steady, uniform, and inviscid conditions

$$
\left(\frac{\partial y}{\partial t} + u\frac{\partial y}{\partial x} + v\frac{\partial y}{\partial y}\right) = -\frac{1}{\rho_m}\frac{\partial p}{\partial y} + y\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) - g \rightarrow p_h = -\rho_m gy
$$

- For Bernoulli equation: Assuming steady and inviscid conditions

$$
\left(\frac{\partial v}{\partial t} + u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y}\right) = -\frac{1}{\rho_m}\frac{\partial p}{\partial y} + \gamma \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) - g \rightarrow \frac{p_b}{\rho_m g} + \frac{V^2}{2g} + y = C
$$

- For Navier-Stokes equation:

$$
\left(\frac{\partial v}{\partial t} + u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y}\right) = -\frac{1}{\rho_m}\frac{\partial p_n}{\partial y} + v_t \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) - g
$$

(FDM with Forward-time and Forward or Backward space scheme is used)

**▶ Velocity calculation**



- Velocity calculations employed the Bubble Image Velocimetry (BIV) method from the PIVlab MATLAB toolbox.
- High-speed videos (120 fps) were used to capture bubble motion. - A QR code provides access to a video demonstration. (Top: Aerated, Bottom: Non-aerated)

#### **▶ Tailwater depth calculation**

- Object Tracking was utilized for fluctuating tailwater depth calculations,

# combining color threshold, edge detection, and the mean-shift tracking algorithm.

- A QR code provides access to a video demonstration. (Top: Aerated, Bottom: Non-aerated)





 $\blacksquare$ 



[Figure 5] Snapshot of velocity vectors (**Left**: Aerated, **Right**: Non-aerated) [Figure 6] Snapshot of Tailwater depth (**Left**: Aerated, **Right**: Non-aerated)

#### **▶ Data acquisition results**

#### **▶ Pressure profile and tailwater depth at downstream wall of the model weir**

Hydrostatio

**Navier Stokes** 

700

600

[Aerated] Comparison of Downstream Pressure Models - Time: 0.00 s

300

400

Pressure (Pa)

500

200

tmospheric pressure

 $0.25$ 

 $0.20$ 

 $50.15$ 

 $0.10$ 

 $0.05$ 

#### **▶ Force acting on the model weir**

Real-time forces obtained from various pressure profiles (Red: Hydrostatic, Green: Bernoulli, Blue: Navier-Stokes).

- Navier-Stokes forces (Blue line) consistently align with measured values (Black line), regardless of aeration or flow rate.



[Figure 9] Contrasting pressure profiles and tailwater depth at downstream wall (Left: Aerated, Right: Non-aerated) [Figure 10] Net force acting on the model weir (Left: low flowrate, Right: high flowrate)



**Goodness of fit**

Average

[Figure 7] Air-Water Pressure, Cavity Pressure, Discharge, and Upstream Water Depth (Left: Aerated, Right: Non-aerated) | [Figure 8] Dimensionless equations for air cavity pressure (Top: Subcritical downstream, Bottom: Sup

